

# Engineers Reference Guide

Warranty

Formulae

Engineers  
Reference  
Guide

Enclosure  
Ratings

Key  
Tables

Conversion  
Factors

Property  
Tables

Nut & Bolts



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## Contents

9.1.	Useful Formulae for Actuator Calculations	2
9.2.	Useful Formulae for Power Transmission Calculations	4
9.3.	Conversion Factors	6
9.4.	Enclosure Ratings	8
9.5.	Metric Nuts and Bolts	9
9.6.	Metric Square and Rectangular Parallel Keys	10
9.7.	Physical Property Values	11
9.8.	Standard SI Prefixes	11
9.9.	Limitations of Responsibility	12
9.10.	Warranty	12



## 9.1. Useful Formulae for Actuator Calculations

### 9.1.1. Metric Units

#### 9.1.1.1. Lifting Screw Lead

Lifting Screw lead (mm) = Screw Pitch (mm) \* Number of Starts on Lifting Screw

#### 9.1.1.2. Calculation of the Raise Per Minute with a Given Worm Shaft Speed

When the worm shaft speed is known, the distance the load can be raised per minute can be determined with this formula:

$$\text{Raise Rate (mm/min)} = \frac{\text{RPM of Worm Shaft} * \text{Lifting Screw Lead (mm)}}{\text{Gear Ratio}}$$

*or alternatively*

$$\text{Raise Rate (mm/min)} = \frac{\text{RPM of Worm Shaft}}{\text{Turns of Worm for 1mm Raise}}$$

#### 9.1.1.3. Calculation of Actuator Input Torque

$$\text{Input Torque (Nm)} = \frac{\text{Load (kN)} * \text{Lifting Screw Lead (mm)}}{2 * \pi * \text{Actuator Efficiency} * \text{Actuator Gear Ratio}}$$

*or alternatively*

$$\text{Input Torque (Nm)} = \frac{\text{Input Power (kW)} * 9550}{\text{Input Speed (rpm)}}$$

#### 9.1.1.4. Calculation of Actuator Input Power

$$\text{Input Power (kW)} = \frac{\text{Load (kN)} * \text{Lifting Screw Lead (mm)} * \text{Input Speed (rpm)}}{60000 * \text{Actuator Efficiency} * \text{Actuator Gear Ratio}}$$

*or alternatively*

$$\text{Input Power (kW)} = \frac{\text{Load (kN)} * \text{Raise Rate (mm/min)}}{60000 * \text{Actuator Efficiency}}$$



## 9.1.2. Imperial Units

### 9.1.2.1. Lifting Screw Lead

Lifting Screw lead (inch) = Screw Pitch (inch) \* Number of Starts on Lifting Screw

### 9.1.2.2. Calculation of the Raise Per Minute with a Given Worm Shaft Speed

When the worm shaft speed is known, the distance the load can be raised per minute can be determined with this formula:

$$\text{Raise Rate (in/min)} = \frac{\text{RPM of Worm Shaft} * \text{Lifting Screw Lead (in)}}{\text{Gear Ratio}}$$

*or alternatively*

$$\text{Raise Rate (in/min)} = \frac{\text{RPM of Worm Shaft}}{\text{Turns of Worm for 1" Raise}}$$

### 9.1.2.3. Calculation of Actuator Input Torque

$$\text{Input Torque (lbf.in)} = \frac{\text{Load (lbf)} * \text{Lifting Screw Lead (inch)}}{2 * \pi * \text{Actuator Efficiency} * \text{Actuator Gear Ratio}}$$

*or alternatively*

$$\text{Input Torque (lbf.in)} = \frac{\text{Input Power (HP)} * 63000}{\text{Input Speed (rpm)}}$$

### 9.1.2.4. Calculation of Actuator Input Power

$$\text{Input Power (HP)} = \frac{\text{Load (lbf)} * \text{Lifting Screw Lead (inch)} * \text{Input Speed (rpm)}}{3.96 * 10^5 * \text{Actuator Efficiency} * \text{Actuator Gear Ratio}}$$

*or alternatively*

$$\text{Input Power (HP)} = \frac{\text{Load (lbf)} * \text{Raise Rate (inch/min)}}{3.96 * 10^5 * \text{Actuator Efficiency}}$$



## 9.2. Useful Formulae for Actuator Calculations

### 9.2.1. Power

	Metric	Imperial
Lifting Motion	$P = \frac{m * g * v}{\eta * 1000}$	$P = \frac{W * v}{\eta * 33000}$
Linear Motion	$P = \frac{F_R * v}{1000}$	$P = \frac{F_R * v}{33000}$
	$F_R = \mu * m * g$	$F_R = \mu * W$
Rotary Motion	$P = \frac{T * n}{9550}$	$P = \frac{T * n}{63000}$

### 9.2.2. Torque

$T = F_R * r$	$T = T * r$
$T = \frac{P * n}{9550}$	$T = \frac{P * n}{63000}$

Symbol	Quantity	Metric Units	Imperial Units
P	Power	kW	HP
T	Torque	Nm	lbf.in
F <sub>R</sub>	Resistance due to Friction	N	lbf
m	Mass	kg	-
W	Weight	-	lb
g	Gravitational Acceleration	9.81 ms <sup>-2</sup>	32.185 ft <sup>-2</sup>
v	Velocity	ms <sup>-1</sup>	ft/min
η	Efficiency	decimals	decimals
μ	Coefficient of Friction	decimals	decimals
n	Rotational Speed	rpm	rpm
r	Radius	m	in



**9.2.3. Moment of Inertia**

	Metric	Imperial
Solid Cylinder	$J = \frac{1}{2} * m * r_{od}^2$	$WK^2 = \frac{1}{2} * W * r_{od}^2$
	$J = \frac{1}{32} * \pi * \rho * l * d_{od}^4$	$WK^2 = \frac{\pi}{32} * r * l * d_{od}^4$
	$J = 0.098 * \rho * l * d_{od}^4$	$WK^2 = 0.1 * \rho * l * d_{od}^4$
Hollow Cylinder	$J = \frac{1}{2} * m * (r_{od}^2 - r_{id}^2)$	$WK^2 = \frac{1}{2} * W * (r_{od}^2 - r_{id}^2)$
	$J = \frac{1}{32} * \pi * \rho * l * (d_{od}^4 - d_{id}^4)$	$WK^2 = \frac{\pi}{32} * \rho * l * (d_{od}^4 - d_{id}^4)$
	$J = 0.098 * \rho * l * (d_{od}^4 - d_{id}^4)$	$WK^2 = 0.1 * \rho * l * (d_{od}^4 - d_{id}^4)$

**9.2.4. Acceleration or Braking Time**

$$t_{acc} = \frac{J * n}{9.55 * T_{acc}} \qquad t_{acc} = \frac{WK^2 * n}{308 * T_{acc}}$$

Symbol	Quantity	Metric Units	Imperial Units
J	Moment of Inertia (metric)	kgm <sup>2</sup>	-
WK <sup>2</sup>	Moment of Inertia (imperial)	-	lb.ft <sup>2</sup>
T <sub>acc</sub>	Torque due to Acceleration or Braking	Nm	lbf.ft
m	Mass	kg	-
W	Weight	-	lb
r <sub>od</sub>	Outer Radius	m	ft
r <sub>id</sub>	Internal Radius	m	ft
d <sub>od</sub>	Outer Diameter	m	ft
d <sub>id</sub>	Internal Diameter	m	ft
l	Length	m	ft
ρ	Density	kg/m <sup>3</sup>	lb/ft <sup>3</sup>
t <sub>acc</sub>	Time for Acceleration or Braking	s	s
n	Rotational Speed	rpm	rpm



### 9.3. Conversion Factors

Length	m	mm	inch	ft
1 m	1	1000	39.370	3.2808
1 mm	0.001	1	0.03937	$3.28 \times 10^{-3}$
1 inch	0.0254	25.4	1	0.0833
1 ft	0.3048	304.8	12	1

Mass	kg	Tonne	lb	Ton (Short)	Ton
1 kg	1	0.001	2.2046	$1.1023 \times 10^{-3}$	$9.842 \times 10^{-4}$
1 Tonne	1000	1	2204.6	1.1023	0.9842
1 lb	0.45355937	$4.536 \times 10^{-4}$	1	$5 \times 10^{-4}$	$4.464 \times 10^{-4}$
1 Ton (Short)	907.185	0.907185	2000	1	0.8929
1 Ton	1016.05	1.016	2240	1.120	1

Force / Weight	N	kgf	kp	lbf
1 N	1	0.1019716	0.120	0.224809
1 kgf	9.80665	1	1	2.2046
1 kp	9.80665	1	1	2.2046
1 lbf	4.44822	0.45359237	0.4536	1

Speed	m/s	mm/s	ft/s	in/s
1 m/s	1	1000	3.2808	39.37
1 mm/s	0.001	1	$3.28 \times 10^{-3}$	0.03937
1 ft/s	0.3048	304.8	1	12
1 in/s	0.0254	25.4	0.0833	1

Torque / Work	Nm	kgf.cm	lbf.in	lbf.ft
1 Nm	1	10.19716	8.8507	0.73756
1 kgf.cm	$9.80665 \times 10^{-2}$	1	0.8679	0.07233
1 lbf.in	0.1129848	1.1521	1	0.08333
1 lbf.ft	1.35582	13.825	12	1



### 9.3. Conversion Factors

Power	kW	Nm/min	kgf.m/s	hp	lbf.ft/min
1 kW	1	60000	10.20	1.34	44220
1 Nm/min	$1.667 \times 10^{-4}$	1	$1.699 \times 10^{-3}$	$2.235 \times 10^{-5}$	0.7374
1 kgf.m/s	$9.807 \times 10^{-3}$	588.6	1	0.01315	433.73
1 hp	0.7457	44741	76.04	1	33000
1 lbf.ft/min	$2.261 \times 10^{-5}$	1.3566	$2.3056 \times 10^{-3}$	$3.03 \times 10^{-5}$	1

Inertia	kg.m <sup>2</sup> (m <sup>2</sup> )	kpms <sup>2</sup>	lbf.ft <sup>2</sup> (WK <sup>2</sup> )	lbf.in <sup>2</sup> (WK <sup>2</sup> )
kg.m <sup>2</sup> (m <sup>2</sup> )	1	0.10197	23.73	3417.2
1 kpms <sup>2</sup>	9.807	1	232.6	33488
1 lbf.ft <sup>2</sup> (WK <sup>2</sup> )	0.0421	$4.30 \times 10^{-3}$	1	144
1 lbf.in <sup>2</sup> (WK <sup>2</sup> )	$2.9264 \times 10^{-4}$	0.6192	$6.944 \times 10^{-3}$	1

Stress / Pressure	MPa (N/mm <sup>2</sup> )	N/m <sup>2</sup>	kg/cm <sup>2</sup>	lbf/inch <sup>2</sup>	lbf/ft <sup>2</sup>
1 MPa (N/mm <sup>2</sup> )	1	$1 \times 10^6$	10.2	145.039	20885.6
1 N/m <sup>2</sup>	$1 \times 10^{-6}$	1	$10.2 \times 10^{-6}$	$145 \times 10^{-6}$	$20.88 \times 10^{-6}$
1 kg/cm <sup>2</sup>	$9.807 \times 10^{-2}$	$9.81 \times 10^3$	1	14.2233	$2.05 \times 10$
1 lbf/inch <sup>2</sup>	$9.8947 \times 10^{-3}$	$6.89 \times 10^3$	0.070307	1	144
1 lbf/ft <sup>2</sup>	$4.7879 \times 10^{-5}$	47.88026	$0.488 \times 10^{-3}$	$6.94 \times 10^{-3}$	1

Temperature	
T °F	$(T \text{ °C} \times 1.8) + 32^\circ$
T °C	$(T \text{ °F} - 32) / 1.8$



## 9.4. Enclosure Ratings

### 9.4.1. IEC Ratings

1st Digit : Solid Ingress		2nd Digit : Liquid Ingress	
0	No special protection	0	No special protection
1	A large surface of the body, such as hand (but no protection against deliberate access.) Solid objects >50 mm diameter.	1	Dripping water (vertically falling drops).
2	Fingers or similar objects not exceeding 80mm in length. Solid objects >12mm in diameter.	2	Vertically dripping water when the enclosure is tilted at any angle up to 15° from its normal position.
3	Tools, wires, etc. of diameter or thickness >2.5mm. Solid objects >1mm diameter.	3	Water falling as a spray at an angle of 60° from the vertical.
4	Wires or strips of thickness >1mm. Solid objects exceeding 1mm diameter.	4	Water splashed against the enclosure from any direction.
5	Ingress of dust is not totally prevented, but dust does not enter in sufficient quantity to interfere with satisfactory operation of the equipment.	5	Water projected by a nozzle against the enclosure.
6	No ingress of dust.	6	Water from heavy seas or projected in powerful jets.
		7	Ingress of water in a harmful quantity not possible when the enclosure is immersed under defined conditions of pressure and time.
		8	Submersible under defined conditions of pressure and time.

### 9.4.2. NEMA and IEC Equivalent Enclosures

Since the IEC degree of protection for enclosed equipment is defined differently from NEMA type enclosure protection, and methods of test are different, exact correlation between IEC IP-type designations and NEMA types is not possible. It is possible to make rough comparisons, which may result in certain applications. The common NEMA type designations compare with IEC designations as follows:

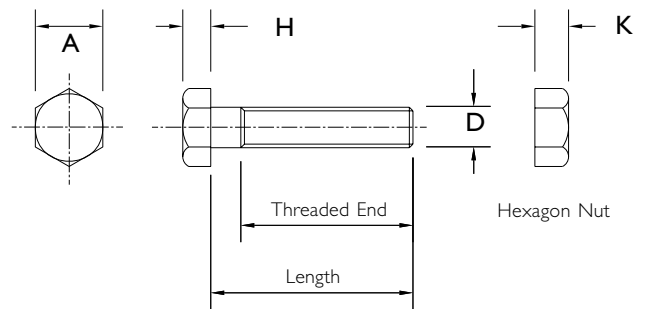
NEMA Type		Nearest IEC Equivalent	
NEMA 1	General protection of people from live parts. Protection against falling dirt. Test: 1/8" to 1/2" (3.175 to 12.7mm) rod entry test and rust resistance tests.	IP2X <sup>①</sup>	Protected against solid objects greater than 12mm. Test: Metallic test finger and 12mm sphere tests. No IEC rust resistance test.
NEMA 3	Dust-tight and sleet resistant. Test: Rain, dust, external icing and rust resistance test.	IP54	Dust protected. Protected against splashing water. Test: Dust and oscillating sprinkler tests. No IEC rust resistance test.
NEMA 3R	Rain-proof and sleet resistant. Test: Rod entry 1/8" to 1/4" (3.175 to 6.35mm), rain, external icing and rust resistance tests.	IP34	Protected against solid objects greater than 2.5mm. Protected against splashing water. Test: 2.5mm rod and oscillating sprinkler tests. No IEC rust resistance or icing tests.
NEMA 4	Water-tight and dust-tight. Test: Hosedown, rust-resistance and external icing tests.	IP65	Dust-tight and protected against water jets. Test: Dust and spray nozzle tests. no rust resistance or external icing test.
NEMA 4X	Water-tight, dust-tight and corrosion resistance. Test: Hosedown, corrosion resistance and external icing tests.	IPW65 <sup>②</sup>	Dust-tight and protected against water jets. Test: Dust and spray nozzle tests. No rust resistance or external icing test.
NEMA 7	Hazardous gas.	-	No IEC equivalent.
NEMA 9	Hazardous dust.	-	No IEC equivalent.
NEMA 12	Dust-tight and drip-tight. Test: Drip, dust and rust resistance tests.	IP6I	Dust-tight and protected against dripping water. Test: Dust and rain simulator tests. No IEC rust resistance test.
NEMA 18	Oil tight and dust tight. Test: Oil-tightness and rust-resistance tests.	IP6X <sup>①</sup>	Dust-tight. Test: Dust test. No IEC oil-tightness or rust resistance tests.

- ① When only one characteristic numeral is used the second numeral is replaced by an 'X'.
- ② A 'W' inserted after the 'IP' indicates suitable for a specified weather condition (conditions and features specified by manufacturer).

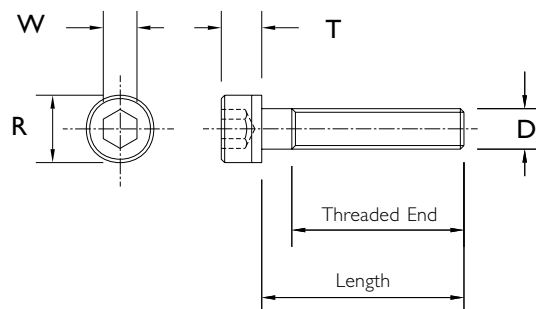
## 9.5. Metric Nuts and Bolts

Thread Size	Pitch	Hexagon Bolts & Nuts			Sockets Head Cap Screw		
		A	H	K	R	T	W
M3	0.50	5.50	2.125	2.40	5.50	3.00	2.5
M4	0.70	7.00	2.925	3.20	7.00	4.00	3.0
M5	0.80	8.00	3.650	4.00	8.50	5.00	4.0
M6	1.00	10.00	4.150	5.00	10.00	6.00	5.0
M8	1.25	13.00	5.650	6.50	13.00	8.00	6.0
M10	1.50	17.00	7.180	8.00	16.00	10.00	8.0
M12	1.75	19.00	8.180	10.00	18.00	12.00	10.0
(M14)	2.00	22.00	9.180	11.00	21.00	14.00	12.0
M16	2.00	24.00	10.180	13.00	24.00	16.00	14.0
(M18)	2.50	27.00	12.215	15.00	27.00	18.00	14.0
M20	2.50	30.00	13.215	16.00	30.00	20.00	17.0
(M22)	2.50	32.00	14.215	18.00	33.00	22.00	17.0
M24	3.00	36.00	15.215	19.00	36.00	24.00	19.0
(M27)	3.00	41.00	17.215	22.00	40.00	27.00	19.0
M30	3.50	46.00	19.620	24.00	45.00	30.00	22.0
(M33)	3.50	50.00	21.260	26.00	50.00	33.00	24.0
M36	4.00	55.00	23.260	29.00	54.00	36.00	27.0
(M39)	4.00	60.00	25.260	31.00	-	-	-
M42	4.50	65.00	26.260	34.00	63.00	42.00	32.0

### Hexagon Bolt



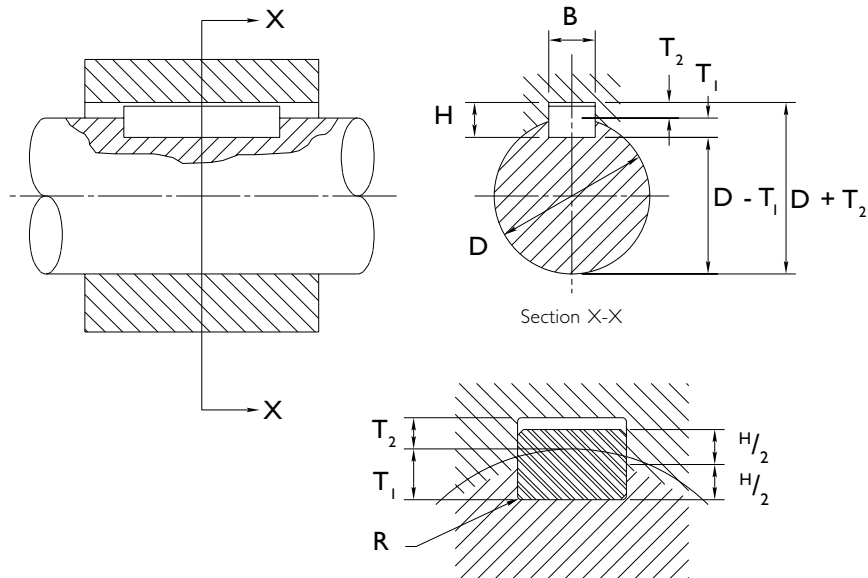
### Socket Head Cap Screw



- Note:**
1. All dimensions in millimetres.
  2. Sizes in brackets are non-preferred standards.
  3. All dimensions are maximum sizes.

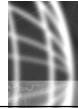
## 9.6. Metric Square and Rectangular Parallel Keys

### Enlarged Detail of Key and Keyways



Symbol		Key B × H width × thick-ness	Keyway												
Nominal Diameter D			Width, B							Depth				Radius, R	
Over	Incl		Nom	Tolerance for class of fit					Shaft, T <sub>1</sub>		Hub, T <sub>2</sub>		Max.	Min.	
				Free		Normal		Close and interference	Nom.	Tol.	Nom.	Tol.			
			Shaft (H9)	Hub (D10)	Shaft (N9)	Hub (Js9)	Shaft and Hub (P9)								
6	8	2 × 2	2	+0.025	+0.060	-0.004	+0.012	-0.006	1.2	+0.1	1.0	+0.1	0.16	0.08	
8	10	3 × 3	3	0	+0.020	-0.029	-0.012	-0.031	1.8	0	1.4	0			
10	12	4 × 4	4	+0.030	+0.078	0	+0.015	-0.012	2.5		1.8				
12	17	5 × 5	5	0	+0.080	-0.030	-0.015	-0.042	3.0		2.3		0.25	0.16	
17	22	6 × 6	6						3.5		2.8				
22	30	8 × 7	8	+0.036	+0.095	0	+0.018	-0.015	4.0	+0.2	3.3	+0.2			
30	38	10 × 8	10	0	+0.040	-0.036	-0.018	-0.051	5.0	0	3.3	0	0.40	0.25	
38	44	12 × 8	12						5.0		3.3				
44	50	14 × 9	14	+0.043	+0.120	0	+0.021	-0.018	5.5		3.8				
50	58	16 × 10	16	0	+0.050	-0.043	-0.021	-0.061	6.0		4.3				
58	65	18 × 11	18						7.0		4.4				
65	75	20 × 12	20						7.5		4.9		0.60	0.40	
75	85	22 × 14	22	+0.052	+0.149	0	+0.026	-0.022	9.0		5.4				
85	95	25 × 14	25	0	+0.065	-0.052	-0.026	-0.074	9.0		5.4				
95	110	28 × 16	28						10.0		6.4				
110	130	32 × 18	32						11.0		7.4				
130	150	36 × 20	36	+0.062	+0.180	0	+0.031	-0.022	12.0	+0.3	8.4	+0.3	1.00	0.70	
150	170	40 × 22	40	0	+0.080	-0.062	-0.031	-0.088	13.0	0	9.4	0			
170	200	45 × 25	45						13.0		10.4				

**Note:** For full range and further information refer BS 4235: Pt 1: 1972

**9.7. Physical Property Values, at 20°C**

Material	Carbon Steel	Aluminium Alloys	Brass 65/35	Copper	Stainless Steel
Density, $\rho$ (kg/m <sup>3</sup> )	7860	2710	8450	8910	7750
Young's Modulus, E (GN/m <sup>2</sup> )	207	710	105	119	190
Shear Modulus, G (GN/m <sup>2</sup> )	79.3	26.2	38	44.7	73.1
Bulk Modulus, K (GN/m <sup>2</sup> )	172	57.5	115	130	178
Poisson's Ratio, $\nu$	0.292	0.334	0.35	0.326	0.305
Coefficient of Thermal Expansion $\times 10^{-6}/K$	12	22	19	17	14
Specific Heat J/kg K	460	920	420	420	460

**Note:** Values given are representative. Exact values may vary with composition and processing, sometimes greatly.

**9.8. Standard SI Prefixes \* †**

Name	Symbol	Factor
exa	E	1 000 000 000 000 000 000 = 10 <sup>18</sup>
peta	P	1 000 000 000 000 000 = 10 <sup>15</sup>
tera	T	1 000 000 000 000 = 10 <sup>12</sup>
giga	G	1 000 000 000 = 10 <sup>9</sup>
mega	M	1 000 000 = 10 <sup>6</sup>
kilo	k	1 000 = 10 <sup>3</sup>
hecto ‡	h	100 = 10 <sup>2</sup>
deca ‡	da	10 = 10 <sup>1</sup>
deci ‡	d	0.1 = 10 <sup>-1</sup>
centi ‡	c	0.01 = 10 <sup>-2</sup>
milli	m	0.001 = 10 <sup>-3</sup>
micro	$\mu$	0.000 001 = 10 <sup>-6</sup>
nano	n	0.000 000 001 = 10 <sup>-9</sup>
pico	p	0.000 000 000 001 = 10 <sup>-12</sup>
femto	f	0.000 000 000 000 001 = 10 <sup>-15</sup>
atto	a	0.000 000 000 000 000 001 = 10 <sup>-18</sup>

\* If possible use multiple and submultiple prefixes in steps of 1000.

† Spaces are used in SI instead of commas to group numbers to avoid confusion with the practise in some European countries of using commas for decimal points.

‡ Not recommended but sometimes encountered.



## Lifting & Positioning Solutions

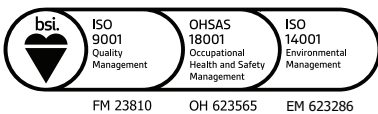
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